



Journal of Computational and Applied Research in Mechanical Engineering Vol. 6, No. 2, pp. 13-26 http://jcarme.srttu.edu



# Influence of heat generation on the phase transformations and impact responses of composite plates with embedded SMA wires

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Article info:	Abstract			
Received: 10/11/2015	In the present research, in contrast to the available papers, not only the			
Accepted: 08/03/2016	superelasticity but also the shape memory effects are taken into account in			
0.1: 02/02/2017	determination of the impact responses. At the same time, in addition to			
Online: $03/03/201/$	modifying Brinson's model for the shape memory alloys (SMAs), to include			
	new parameters and loading events, and Hertz contact law, distributions of			
Keywords:	the SMA phases are considered to be both localized and time-dependent. Furthermore, effects of the impact-induced heat generation and mechanical energy on the resulting histories of the martensite phase volume fraction,			
Shape memory alloy				
-				
Impact,	stress-strain, temperature, lateral deflection, and contact force are			
Phase transformation,	investigated. The generated heat in the SMA wires during the impact is determined through using a Helmholtz free energy function including the latent heat of the phase transformation. The resulting governing equations are			
Heat generation,				
Hybrid composite plate				
nijena compense place.	solved by the finite element method. The nonlinear refined constitutive laws			
	are solved through a return-mapping Newton-Raphson procedure. Results			
	reveal that incorporation of the heat generation effects is significant in			
	medium/high-velocity impacts or when the stress field is almost uniform			

### 1. Introduction

In many cases, the structural damping does not render adequately dissipation of the energies of the undesired structural oscillations. Attenuation of the vibration amplitude may be accomplished through using the SMA elements that consume the undesired energy for the phase transformations of the microstructure of the material. Some limited authors have studied low velocity impact responses of composite plate with embedded SMA wires. Khalili et al. [1], Shokuhfar et al. [2], Meo et al. [3], and Kim et al. [4] are among the well-known researchers

\*Corresponding author email address: shariyat@kntu.ac.ir in this area. Roh and Kim studied low-velocity impact responses [5] of the hybrid smart composite plates, employing a finite element formulation based on the first-order sheardeformation plate theory. The majority of the numerical investigations on low-velocity impact of the SMA composite plates have been accomplished either by using the recovery stress concept or assumption of a uniform distribution of the martensite phase distribution. Furthermore, effects of the lateral flexibility and diversity of the materials of the underneath layers have not been considered in the majority of the available articles. Shariyat and Jafari [6] and Shariyat and Farzan Nasab [7, 8] proposed modified contact laws that overcome these shortcomings. Using outcomes of these researches and modifying Brinson's model, Shariyat and Moradi [9] investigated impact responses of hybrid composite plates with embedded SMA wires. Recently, Shariyat and Hosseini [10] have extended these novelties through accurate eccentric impact analysis of the preloaded SMA composite plates, proposing a new mixed-order hyperbolic global-local plate theory.

Effects of the loading rate on responses of the abruptly loaded SMA structures were discussed by some authors. In order to simulate the strain rate effect in phase transformation of the SMA, Helm and Haupt [11] proposed a model within the continuum thermo-mechanics framework and showed that the SMAs exhibit strong thermo-mechanical coupling and consequently, the material model must comply with the second law of thermodynamic. Tobushi et al. [12] studied the influence of the load-rate on superelastic properties of the TiNi shape memory alloy and demonstrated that the temperature increases in proportion to the strain rate. Kadkhodaei et al. [13] presented a Helmholtz free energy expression for the SMAs that involved effects of the strain rate. Other studies in this field have been published by Monteiro et al. [14], Morin et al. [15], and Roh [16].

All the aforementioned impact researches have only considered effects of the superelastic characteristics of the SMA wires on the impact responses. In the present analysis, effects of the superelasticity and shape memory natures of the SMA wires of the impacted SMA composite plates are considered simultaneously, for the first time. In addition to refining Brinson's model for the SMAs and Hertz contact law, influences of the impact-induced-heat on the resulting phase transformations and responses are considered based on the energy balance and thermodynamic equations. The finite element form of the governing equations is solved by an iterative numerical time integration method wherein the nonlinear constitutive equations are solved by a Newton-Raphson procedure, within each time step.

#### 2. The governing equations

#### 2.1. The thermodynamic formulation

Length, width, and thickness of the plate are denoted by a, b, and h, respectively and initial velocity and radius of the rigid spherical indenter are denoted by V and R, respectively (Fig. 1).



**Fig. 1.** The configuration of the composite plate with embedded SMA wires, and the employed rectangular elements.

Effects of transformation rate of the SMA wires on the resulting temperature rise may be determined based on the first law of thermodynamics [17, 18]:

$$q_{i,i} - \sigma_{ij}\dot{\varepsilon}_{ij} + \rho_0 (\dot{f} + \dot{T}S)$$
(1)  
-  $\rho_0 (r - T\dot{S}) = 0$ 

where  $q, \sigma_{ij}, \dot{\varepsilon}_{ij}, \rho_0, f, T, S$ , and r are the heat flux vector, Cauchy's stress tensor, strain-rate tensor, mass density, Helmholtz free energy, temperature, entropy, and the internal heat generation, respectively. The comma symbol denotes a partial differentiation with respect to the indicated parameter. On the other hand, based on the second law of thermodynamics:

$$\sigma_{ij} = \rho_0 \frac{\partial f}{\partial \varepsilon_{ij}}$$

$$S = -\frac{\partial f}{\partial T}$$

$$\Lambda = -\rho_0 \frac{\partial f}{\partial \varepsilon}$$
(2)

where  $\Lambda$  is the driving force of the transformation and  $\xi$  is the martensite volume fraction. The Helmholtz free energy expression that is especially suitable for dynamic loading

of SMAs may be proposed by extending that of Ref. [13]:

$$f = \frac{C_{ijkl}\varepsilon_{ij}\varepsilon_{kl}}{2\rho_0} - \frac{\beta_{ij}(T - T_0)\varepsilon_{ij}}{\rho_0} - \frac{\varepsilon_l\xi_sE_sV_s\varepsilon_{11}}{\rho_0} + V_s\frac{\lambda}{T^*}(T - T^*)\xi \qquad (3) + C_E\left(T - T_0 - Tln\left(\frac{T}{T_0}\right)\right)$$

where  $C_{ijkl}$ ,  $\beta_{ij}$ ,  $\varepsilon_l$ ,  $\xi_s$ ,  $E_s$ ,  $V_s$ ,  $\lambda$ ,  $T^*$ ,  $C_E$ , and  $T_0$ are the elasticity tensor, stress-temperature coupling coefficients, maximum recoverable strain, stress-induced martensite volume fraction, modulus of the SMA, SMA volume fraction of the hybrid composite, latent heat of the phase transformation, critical temperature, the specific heat of the material, and initial temperature, respectively. Comparing Eqs. (2) and (3), the transformation driving force and entropy may be written in the following explicit forms:

$$\Lambda = -\rho_0 \frac{\partial f}{\partial \xi} = \varepsilon_l E_s k_s \varepsilon_{11} - \rho_0 V_s \frac{\lambda}{T^*} (T - T^*)$$
<sup>(4)</sup>

$$S = -\frac{\partial f}{\partial T} = \frac{\beta_{ij}\varepsilon_{ij}}{\rho_0} - V_s \frac{\lambda}{T^*}\xi + C_E ln\left(\frac{T}{T_0}\right)$$
(5)

The second term of the last side of Eq. (5), i.e.,  $V_s \frac{\lambda}{T^*} \xi$  that has been ignored by other researchers is retained in the present research. The rate form of the entropy equation (5) may be written as follows:

$$\dot{S} = \frac{\beta_{ij}\dot{\varepsilon}_{ij}}{\rho_0} - V_s \frac{\lambda}{T^*} \dot{\xi} + C_E \frac{\dot{T}}{T}$$
(6)

Generally, in low-velocity impacts, the first term of right-hand side of Eq. (6), i.e.,  $\frac{\beta_{ij} \varepsilon_{ij}}{\rho_0}$  may be neglected in comparison to the second one,

that contains the heat generated by the latent heat [17, 18].

$$q_{i,i} - (\varepsilon_l E_s \varepsilon_{11} + \rho_0 \lambda) V_s \dot{\xi} + \rho_0 C_E \dot{T} = 0$$
(7)

Equation (7) reveals that the martensite transformation produces heat and causes temperature rise and vice versa.

#### 2.2. The refined constitutive and contact laws

The constitutive relation of the *i*th layer of the plate in the principal coordinates of the material is:

$$\begin{cases} \sigma_{1} \\ \sigma_{2} \\ \tau_{12} \\ \tau_{13} \\ \tau_{23} \end{cases}$$

$$= \begin{bmatrix} Q_{11} & Q_{12} & 0 & 0 & 0 \\ Q_{12} & Q_{22} & 0 & 0 & 0 \\ 0 & 0 & Q_{44} & 0 & 0 \\ 0 & 0 & 0 & Q_{55} & 0 \\ 0 & 0 & 0 & 0 & Q_{66} \end{bmatrix}$$

$$\begin{cases} \varepsilon_{1} - \alpha_{1} \Delta T \\ \varepsilon_{2} - \alpha_{2} \Delta T \\ \gamma_{12} \\ \gamma_{13} \\ \gamma_{23} \end{cases} = \boldsymbol{Q} (\boldsymbol{\varepsilon} - \boldsymbol{\varepsilon}_{T})$$

where the subscripts 1, 2, and 3 correspond to the fiber direction, the in-plane transverse direction of the fiber and the out-of plane transverse direction of the fiber, respectively, and the non-zero components of the reduced stiffness matrix Q are:

$$Q_{11} = \frac{E_1}{1 - v_{12}v_{21}}$$

$$Q_{22} = \frac{E_2}{1 - v_{12}v_{21}}$$

$$Q_{12} = \frac{v_{12}E_2}{1 - v_{12}v_{21}}, \quad Q_{44}$$

$$= G_{12}$$

$$Q_{55} = G_{13}, \quad Q_{66} = G_{23}$$
(9)

where E, v, G,  $\alpha$ , and  $\varepsilon_T$  are Young's modulus, Poisson's ratio, shear modulus, the coefficient of the thermal expansion, and vector of the thermal strains, respectively. Based on Brinson's constitutive equation of the shape memory alloy [19], the stress and strain quantities may be related through the martensite volume fraction ( $\xi$ ) as:

$$\sigma - \sigma_0 = E(\xi)\varepsilon - E(\xi_0)\varepsilon_0 + \Omega(\xi)\xi_s - \Omega(\xi_0)\xi_{s0} + \Theta(T - T_0),$$

$$\xi = \xi_s + \xi_T$$
(10)

where  $\Omega$  is the transformation function and  $\xi_s$ and  $\xi_T$  represent the stress- and temperatureinduced martensite volume fractions, respectively. The subscript 0 denotes the initial quantities of the local or global event under investigation. The Young's modulus of elasticity of the SMA is dependent on the martensite and austenite moduli; so that, according to the rule of mixtures:

$$E(\xi) = E_A + \xi(E_M - E_A)$$
  

$$\Omega(\xi) = -\varepsilon_l E(\xi)$$
(11)

where  $E_A$  and  $E_M$  are Young's moduli of the austenite and martensite phases, respectively. In this regard, the following kinetic laws of the transformation may be used:

(i) Conversion from the austenite to the detwinned martensite phase  $(\sigma_s^{cr} + C_M(T - M_s) < \sigma < \sigma_f^{cr} + C_M(T - M_s)$  and  $T > M_s$ ):

$$\xi_{s} = \frac{1 - \xi_{s0}}{2} \times \cos\left\{\frac{\pi}{\sigma_{s}^{cr} - \sigma_{f}^{cr}} \left[\sigma\right] - \sigma_{f}^{cr} - C_{M}(T - M_{s})\right] + \frac{1 + \xi_{s0}}{2}$$
(12)  
$$\xi_{T} = \xi_{T0} - \frac{\xi_{T0}}{1 - \xi_{s0}} (\xi_{s} - \xi_{s0})$$

(ii) Conversion from de-twinned martensite to austenite phase. For  $T > A_s$  and  $C_A(T - A_f) < \sigma < C_A(T - A_s)$ 

$$\xi = \frac{\xi_0}{2} \times \left\{ \cos \left[ \frac{\pi}{A_f - A_s} \left( T - A_s - \frac{\sigma}{C_A} \right) \right] + 1 \right\}$$
(13)  
$$\xi_s = \xi_{s0} - \frac{\xi_{s0}}{\xi_0} (\xi_0 - \xi),$$
  
$$\xi_T = \xi_{T0} - \frac{\xi_{T0}}{\xi_0} (\xi_0 - \xi)$$

 $A_s, A_f$ , and  $M_s$  correspond to the start and finish temperatures of the austenite phase transformation and temperature of the start of the martensite transformation, respectively.  $C_M, C_A, \sigma_s^{cr}$ and  $\sigma_f^{cr}$ are the **SMA** transformation properties [20]. Since the structure is subjected to thermo-mechanical loads, Eq. (8) may be modified for a layer with embedded SMA wires in the transformed coordinates, as:

$$\begin{split} \overline{\sigma} &= \begin{cases} \sigma_{xx} \\ \sigma_{yy} \\ \tau_{xy} \\ \tau_{xz} \\ \tau_{yz} \end{cases} \\ \\ = \begin{bmatrix} \overline{Q}_{11} & \overline{Q}_{12} & \overline{Q}_{16} & 0 & 0 \\ \overline{Q}_{12} & \overline{Q}_{22} & \overline{Q}_{26} & 0 & 0 \\ \overline{Q}_{16} & \overline{Q}_{26} & \overline{Q}_{66} & 0 & 0 \\ 0 & 0 & 0 & \overline{Q}_{45} & \overline{Q}_{55} \\ 0 & 0 & 0 & \overline{Q}_{44} & \overline{Q}_{45} \end{bmatrix} \times \\ & \left\{ \begin{cases} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \\ \gamma_{xz} \\ \gamma_{yz} \end{cases} - \begin{cases} \alpha_{xx} \Delta T \\ \alpha_{yy} \Delta T \\ 2\alpha_{xy} \\ 0 \\ 0 \end{cases} \right\} \\ \\ - \begin{cases} k_s E_s \xi_s \varepsilon_L \cos^2(\theta) \\ k_s E_s \xi_s \varepsilon_L \sin^2(\theta) \\ k_s E_s \xi_s \varepsilon_L \sin^2(\theta) \\ k_s E_s \xi_s \varepsilon_L \sin(\theta) \cos(\theta) \\ 0 \\ 0 \end{bmatrix} \\ \\ = \overline{Q}(\overline{\varepsilon} - \overline{\varepsilon}_T) - \overline{\sigma}^s \end{split}$$
(14)

where,  $E_s$  and  $\overline{\epsilon}_T$  are the modulus of elasticity of the SMA material and thermal strains vector in the body coordinates of the material, respectively. The non-zero components of the transformed reduced stiffness matrix  $\overline{Q}$  are [21]:

$$\overline{Q}_{11} = Q_{11}cos^{4}\theta + 2(Q_{12} + 2Q_{66})sin^{2}\theta cos^{2}\theta + Q_{22}sin^{4}\theta \overline{Q}_{12} = (Q_{11} + Q_{22} - 4Q_{66})sin^{2}\theta cos^{2}\theta + Q_{12}(sin^{4}\theta + cos^{4}\theta) \overline{Q}_{22} = Q_{11}sin^{4}\theta + 2(Q_{12} + 2Q_{66})sin^{2}\theta cos^{2}\theta + Q_{22}cos^{4}\theta \overline{Q}_{16} = (Q_{11} - Q_{12} - 2Q_{66})sin^{3}\theta cos\theta + (Q_{12} - Q_{22} + 2Q_{66})sin^{3}\theta cos\theta + (Q_{12} - Q_{22} + 2Q_{66})sin^{2}\theta cos^{2}\theta + Q_{66}(sin^{4}\theta + cos^{4}\theta) \overline{Q}_{44} = Q_{44}cos^{2}\theta + Q_{55}sin^{2}\theta \overline{Q}_{45} = (Q_{55} - Q_{44})cos\theta sin\theta \overline{Q}_{55} = Q_{55}cos^{2}\theta + Q_{44}sin^{2}\theta a_{xx} = a_{1}cos^{2}\theta + a_{2}cos^{2}\theta a_{yy} = (a_{1} - a_{2})sin\theta cos\theta$$

Since it is assumed that the SMA wires are parallel to the composite fibers, the following relations may be used to determine the effective material properties of each layer in the principal directions of the fibers, using the micromechanical relations appeared in Eq. (11) [22]:

$$E_{1} = E_{1c}V_{c} + E_{s}V_{s}$$

$$E_{2} = E_{2c}E_{s}/(V_{c}E_{s} + V_{s}E_{2c})$$

$$G_{12} = G_{12c}G_{s}/(V_{c}G_{s} + V_{s}G_{12c})$$

$$G_{13} = G_{13c}G_{s}/(V_{c}G_{s} + V_{s}G_{13c})$$

$$v_{12} = v_{12c}V_{c} + v_{s}V_{s}$$

$$\rho = \rho_{c}V_{c} + \rho_{s}V_{s}$$

$$\alpha_{1} = \alpha_{1c}V_{c} + \alpha_{s}V_{s}$$

$$\alpha_{2} = \alpha_{2c}\alpha_{s}/(\alpha_{2c}V_{s} + \alpha_{s}V_{c})$$
(16)

where the subscripts s and c stand for the SMA and composite materials, respectively. The traditional Hertz contact law should be modified based on a proper micromechanical model and a further refinement that considers the finite thickness of the plate is required. For a transversely isotropic plate impacted by a rigid spherical indenter, Turner's modification of Hertz law [23] may be further enhanced based on the employed micromechanical model, as follows [24]:

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5)

$$\frac{\frac{8}{3}G_{12}\sqrt{2R}\hat{\alpha}^{3/2}}{(1-\nu_{12})\sqrt{\frac{E_{1}/E_{2}-\nu_{13}^{2}}{1-\nu_{12}^{2}}}\sqrt{\sqrt{\frac{E_{1}/E_{2}-\nu_{13}^{2}}{1-\nu_{12}^{2}}} + \frac{(E_{1}/2G_{13})-\nu_{13}(1+\nu_{12})}{1-\nu_{12}^{2}}} \quad (17)$$

where,  $\hat{\alpha}$  is the indentation value. For the unloading phase, Yang and Sun law [25] may be employed:

$$F_c = F_{max} \left( \frac{\hat{\alpha} - \hat{\alpha}_0}{\hat{\alpha}_{max} - \hat{\alpha}_0} \right)^{\frac{5}{2}}$$
(18)

where,  $F_{max}$  is the maximum contact force reached during the impact,  $\alpha_{max}$  is the maximum indentation corresponds to  $F_{max}$  and  $\alpha_0$  is the permanent indentation, if any.

# 2.3. The governing nonlinear equations of motion

Using the quadrilateral element, the finite element form of Reddy's third-order shear deformation theory becomes:

where  $\boldsymbol{\delta}$  and  $\boldsymbol{\Delta}$  are the displacement components and nodal displacement components vectors, respectively. U, V, and W are the nodal displacements of the reference plane, and  $\boldsymbol{\Phi}_x$ and  $\boldsymbol{\Phi}_y$  are nodal rotations of the cross-section in the x-z and y-z planes, respectively.  $\boldsymbol{\mathcal{N}}$  is the shape functions matrix of the quadrilateral element [26]. Therefore:

$$\bar{\boldsymbol{\varepsilon}} = \begin{cases} \boldsymbol{\varepsilon}_{x} \\ \boldsymbol{\varepsilon}_{y} \\ \boldsymbol{\gamma}_{xy} \\ \boldsymbol{\gamma}_{yz} \end{cases} = \begin{bmatrix} \frac{\partial}{\partial x} & 0 & 0 \\ 0 & \frac{\partial}{\partial y} & 0 \\ \frac{\partial}{\partial y} & \frac{\partial}{\partial x} & 0 \\ \frac{\partial}{\partial z} & 0 & \frac{\partial}{\partial x} \\ 0 & \frac{\partial}{\partial z} & \frac{\partial}{\partial y} \end{bmatrix} \boldsymbol{\delta}$$
(20)
$$= \boldsymbol{\mathcal{D}} \mathbb{N} \boldsymbol{\Delta} = \boldsymbol{\mathcal{B}} \boldsymbol{\Delta}$$

The transverse distribution of the temperature rise may be assumed to linear; an assumption that may be justified, especially for the thin plates (as using the first-order plate theory):

$$\theta^{*}(\mathbf{x}, \mathbf{y}, \mathbf{z}, \mathbf{t}) = \mathbf{T}_{0}(\mathbf{x}, \mathbf{y}, \mathbf{t}) + \mathbf{z}\mathbf{T}_{1}(\mathbf{x}, \mathbf{y}, \mathbf{t}) = [\mathcal{N} \quad \mathbf{z}\mathcal{N}] \begin{Bmatrix} \mathbf{T}_{0} \\ \mathbf{T}_{1} \end{Bmatrix}$$
(21)  
=  $\mathbf{G}\Theta$ 

The governing equations may be derived using the variational principles. Based an approach proposed earlier by Serra and Bonaldi [27] (not for SMAs) the residues of the energy balance Eq. (7) and the equation of motion in terms of the stress components may be multiplied respectively, by variations of the temperature and displacement components in the integrals of the weighted residuals. Combining the two equations, using Fourier heat transfer law and integrating by parts, leads to:

$$\int_{V} \rho_{0} C_{E} \dot{\theta}^{*} \delta \theta^{*} dV 
- \int_{V} \varepsilon_{l} E_{s} V_{s} \varepsilon_{11} \dot{\xi} \delta \theta^{*} dV 
- \int_{V} \rho_{0} V_{s} \lambda \dot{\xi} \delta \theta^{*} dV 
+ \int_{V} k_{ij} \theta^{*}_{,j} \delta \theta^{*}_{,i} dV$$

$$+ \int_{V} \sigma_{ij} \delta \bar{\varepsilon}_{ij} dV + \int_{V} \rho \ddot{u}_{i} \delta u_{i} dV 
= - \int_{A} q_{i} n_{i} \delta \theta^{*} dA 
+ \int_{A} \sigma_{ij} n_{j} \delta u_{i} dA + \int_{V} X_{i} \delta u_{i} dV$$
(22)

The proposed terms of Eq. (22) should be expanded and integrated through the thickness. Substituting the quantities by their finite element equivalents, leads to the following result, in absence of the body forces:

$$\delta \boldsymbol{\Delta}^{T} \int_{\Omega_{0}} \int_{-\frac{h}{2}}^{\frac{h}{2}} (\rho_{0} \mathbb{N}^{T} \mathbb{N} \boldsymbol{\ddot{\Delta}} + \boldsymbol{B}^{T} \overline{\boldsymbol{Q}} \boldsymbol{B} \boldsymbol{\Delta}) dz d\Omega_{0} \\ + \delta \boldsymbol{\Theta}^{T} \left\{ \int_{\Omega_{0}} \left[ \int_{-\frac{h}{2}}^{\frac{h}{2}} ([\rho_{0} C_{E} \boldsymbol{G}^{T} \boldsymbol{G} \boldsymbol{\dot{\Theta}} + k_{ij} \boldsymbol{G}_{,i}^{T} \boldsymbol{G}_{,j} \boldsymbol{\Theta}] - \varepsilon_{l} E_{s} V_{s} \dot{\boldsymbol{\xi}} \boldsymbol{G}^{T} [1 \quad 0 \quad 0 \quad 0 \quad 0] \boldsymbol{B} \boldsymbol{\Delta}) dz \\ + h \boldsymbol{G}^{T} \boldsymbol{G} \boldsymbol{\Theta} \right] d\Omega_{0} \right\}$$
(23)  
$$= \delta \boldsymbol{\Theta}^{T} \int_{\Omega_{0}} \left\{ \int_{-\frac{h}{2}}^{\frac{h}{2}} \rho_{0} V_{s} \lambda \dot{\boldsymbol{\xi}} \boldsymbol{G}^{T} dz - [q_{n} + h(T_{initial} - T_{\infty})] \boldsymbol{G}^{T} \right\} d\Omega_{0} \\ + \delta \boldsymbol{\Delta}^{T} \int_{\Omega_{0}} \int_{-\frac{h}{2}}^{\frac{h}{2}} \boldsymbol{B}^{T} (\overline{\boldsymbol{Q}} \boldsymbol{\varepsilon}_{T} + \boldsymbol{\sigma}^{s}) dz d\Omega_{0} \\ + F_{c} \delta w_{i}$$

where  $\Omega_0$  represents area of the reference plane of the plate and the subscripts, n and s denote directions normal and tangent to the boundary, respectively.  $F_c$  and  $w_i$  are respectively the contact force and vertical displacement of the indenter and  $q_n$  denotes the transverse thermal flow. In the derivation of Eq. (23), the total transverse thermal heat flux is assumed to include the conduction and convection heat fluxes:

$$Q_n = q_n + h(\mathbf{G}\mathbf{\Theta} + T_{initial} - T_{\infty})$$
(24)

where, h is coefficient of the convection heat transfer of the top/bottom surface of the plate. The temperature rise and displacement parameters may be gathered in an augmented vector of unknowns as follows:

$$\begin{cases} u \\ v \\ w \\ \theta^* \end{cases} = \begin{bmatrix} \mathbb{N} & \mathbf{0} \\ \mathbf{0} & \mathbf{G} \end{bmatrix} \begin{bmatrix} \mathbf{\Delta} \\ \mathbf{\Theta} \end{bmatrix} = \Xi \mathbf{\Lambda}$$
 (25)

Based on Eq. (23), the coupled governing thermomechanical equations of motion of the plate became:

$$\delta \Lambda^{T} \Big[ \mathcal{M} \ddot{\Lambda} + \mathcal{C}(\Lambda) \dot{\Lambda} + \mathcal{K}(\Lambda) \Lambda \\ - \mathcal{F}(\Lambda) \Big] = 0$$
(26)

Equation (26) must hold for any arbitrary choice of  $\delta \Lambda \neq 0$ . Therefore,

$$\mathcal{M}\ddot{\Lambda} + \mathcal{C}(\Lambda)\dot{\Lambda} + \mathcal{K}(\Lambda)\Lambda = \mathcal{F}(\Lambda)^T \qquad (27)$$

The system of Eq. (27) has to be assembled with the governing equation of motion of the indenter based on Newton's second law [28, 29]:

 $m_i \ddot{w}_i + k_c (w_i - w_c)^{3/2} = 0$  (28) where  $k_c$  and  $w_c$  are the impact stiffness and lateral deflection of the central point of the

plate, respectively.

#### 3. Solution procedure

The resulting governing finite element equations of the hybrid plate are highly

nonlinear; because both nonlinear piecewisedefined material constitutive equations and nonlinear contact law are used. The resulting system of equations is solved iteratively within each time step, using a return-mapping Newton-Raphson algorithm that is especially used to solve the non-linear constitutive equation of the SMA. In this procedure, the residue may be defined based on Eq. (10) as:

$$\mathcal{R} = \sigma - E(\xi)\varepsilon - E(\xi_0)\varepsilon_0 + \Omega(\xi)\xi_s - \Omega(\xi_0)\xi_{s0} + \Theta(T - T_0) = \mathbf{K}(\sigma)\sigma - \mathbf{F}$$
(28)

So that the tangent stiffness matrix  $(K_T)$  and increment of the unknown stress of an arbitrary nodal point may be defined as follows:

$$K_T = \frac{\partial \mathcal{R}}{\partial \sigma}\Big|_{\sigma}, \qquad \Delta \sigma = -K_T^{-1} \mathcal{R}$$
 (29)

Therefore, based on Eqs. (11), and (14):  $\mathcal{D} = \overline{\mathcal{Q}} (c - c - AT)$ 

$$\mathcal{R} = Q_{11}(\varepsilon_{x} - \alpha_{xx}\Delta I) + \bar{Q}_{12}(\varepsilon_{yy} - \alpha_{yy}\Delta T) + \bar{Q}_{16}(\gamma_{xy} - 2\alpha_{xy}\Delta T) - V_{s}E_{s}\xi_{s}\varepsilon_{l}\cos^{2}\theta - \sigma_{x}$$

$$K_{T} = \frac{\partial\bar{Q}_{11}}{\partial\sigma}(\varepsilon_{x} - \alpha_{x}\Delta T) + \bar{Q}_{11}\left(-\frac{\partial\alpha_{x}}{\partial\sigma}\Delta T\right) + \frac{\partial\bar{Q}_{12}}{\partial\sigma}(\varepsilon_{y} - \alpha_{y}\Delta T) + \bar{Q}_{12}\left(-\frac{\partial\alpha_{y}}{\partial\sigma}\Delta T\right) + \frac{\partial\bar{Q}_{16}}{\partial\sigma}(\gamma_{xy} - 2\alpha_{xy}\Delta T) + \bar{Q}_{16}\left(-2\frac{\partial\alpha_{xy}}{\partial\sigma}\Delta T\right) + \bar{Q}_{16}\left(-2\frac{\partial\alpha_{xy}}{\partial\sigma}\Delta T\right) - V_{s}\varepsilon_{l}\cos^{2}\theta\frac{\partial(E_{s}\xi_{s})}{\partial\sigma} - 1$$

$$(30)$$

The stress value of the *k*th iteration may be determined from:

$$\sigma^{k+1} = \sigma^k + \Delta \sigma \tag{31}$$

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The expanded form of the partial differentiations appeared in Eq. (30) is:

$$\frac{\partial \bar{Q}_{11}}{\partial \sigma} = \frac{\partial Q_{11}}{\partial \sigma} \cos^4 \theta + 2 \left( \frac{\partial Q_{12}}{\partial \sigma} + 2 \frac{\partial Q_{44}}{\partial \sigma} \right) \sin^2 \theta \cos^2 \theta + \frac{\partial Q_{22}}{\partial \sigma} \sin^4 \theta \frac{\partial \bar{Q}_{12}}{\partial \sigma} = \left( \frac{\partial Q_{11}}{\partial \sigma} + \frac{\partial Q_{22}}{\partial \sigma} - 4 \frac{\partial Q_{44}}{\partial \sigma} \right) \sin^2 \theta \cos^2 \theta + \frac{\partial Q_{12}}{\partial \sigma} (\sin^4 \theta + \cos^4 \theta) \frac{\partial \alpha_{xx}}{\partial \sigma} = \frac{\partial \alpha_1}{\partial \sigma} (\cos \theta)^2 + \frac{\partial \alpha_2}{\partial \sigma} (\sin \theta)^2, \quad \frac{\partial \alpha_{yy}}{\partial \sigma} \quad (32) = \frac{\partial \alpha_1}{\partial \sigma} (\sin \theta)^2 + \frac{\partial \alpha_2}{\partial \sigma} (\cos \theta)^2 \frac{\partial \bar{Q}_{16}}{\partial \sigma} = \left( \frac{\partial Q_{11}}{\partial \sigma} - \frac{\partial Q_{12}}{\partial \sigma} - 2 \frac{\partial Q_{44}}{\partial \sigma} \right) \sin \theta \cos^3 \theta + \left( \frac{\partial Q_{12}}{\partial \sigma} - \frac{\partial Q_{22}}{\partial \sigma} + 2 \frac{\partial Q_{44}}{\partial \sigma} \right) \sin \theta \cos \theta \\\frac{\partial \alpha_{xy}}{\partial \sigma} = \left( \frac{\partial \alpha_1}{\partial \sigma} - \frac{\partial \alpha_2}{\partial \sigma} \right) \sin \theta \cos \theta \\\frac{\partial (E_s \xi_s)}{\partial \sigma} = \frac{\partial \xi}{\partial \sigma} (E_M - E_A) \xi_s + \frac{\partial \xi_s}{\partial \sigma} E_s \\ and based on Exercise (10) and (13);$$

- and, based on Eqs. (12) and (13):(i) For conversion from the austenite to the
  - de-twinned martensite phase:

$$\frac{\partial \xi_{s}}{\partial \sigma} = \frac{1 - \xi_{s0}}{2} \left[ \frac{-\pi}{\sigma_{s}^{cr} - \sigma_{f}^{cr}} \sin \left\{ \frac{\pi}{\sigma_{s}^{cr} - \sigma_{f}^{cr}} \left[ \sigma \right] \right\} \right]$$

$$- \sigma_{f}^{cr} - C_{M}(T - M_{s}) \right] \right\}$$

$$\frac{\partial \xi_{T}}{\partial \sigma} = -\frac{\xi_{T0}}{1 - \xi_{s0}} \frac{\partial \xi_{s}}{\partial \sigma}, \quad \frac{\partial \xi}{\partial \sigma} = \frac{\partial \xi_{s}}{\partial \sigma} + \frac{\partial \xi_{T}}{\partial \sigma}$$
(33)

(ii) For conversion from de-twinned martensite to austenite phase:

$$\frac{\partial \xi}{\partial \sigma} = \frac{\xi_0}{2} \frac{\pi}{(A_f - A_s)C_A} \sin\left[\frac{\pi}{A_f - A_s} \left(T - A_s - \frac{\sigma}{C_A}\right)\right] \\
- A_s - \frac{\sigma}{C_A}\right] \\
\frac{\partial \xi_s}{\partial \sigma} = \frac{\xi_{s0}}{\xi_0} \frac{\partial \xi}{\partial \sigma}$$
(34)

#### 4. Results and discussion

While the element size is adopted based on a convergence analysis (whose results are not included here due to length restriction of the paper) in the present research, an integration time step in the order of  $10^{-5}$ (sec) is adopted to accurately trace the time history of the displacement parameters. This time step is much less than the fundamental period time of the structure and especially much less than the response time of the structure.

#### 4.1. Verification of the results

Since instantaneous and localized phase transformations of the SMA wires have not been considered in the limited available researches on the impact analysis of the hybrid SMA composite plates, an example previously presented by Tiberkak et al. [30] is reconsidered to verify the results.



**Fig. 2.** A comparison between present and Tiberkak et al. [30] results for the time histories of the contact force.

The predicted time histories of the contact force and lateral deflection predicted by the present analysis are compared with those of Tiberkak et al. in Figs. 2 and 3, respectively. These comparisons confirm the good agreement between the results.

#### 4.2. Results of the parametric study

In order to investigate effects of the impactinduced heat generation on the impact responses of composite plates with SMA wires, a simply supported plate whose top and bottom surfaces have convection heat transfer with the ambient is considered. Length, width, and thickness of the plate are 0.100, 0.100, and 0.005 m, respectively. Specifications of the [0/90/0/90/0] composite plate and the indenter are as follows:

 $\begin{array}{ll} \mbox{Plate:} & E_1 = 120.7 \mbox{ GPa}, E_2 = 7.93 \mbox{ GPa}, G_{12} \\ = G_{13} = 5.5 \mbox{ GPa}, \nu_{12} = 0.3, \rho = 1580 \frac{\mbox{kg}}{\mbox{m}^3} \\ \mbox{Indenter:} & E = 207 \mbox{ GPa}, \nu = 0.3 \end{array}$ 

To choose the proper analysis conditions for the SMA wires, the three loading paths shown in Fig. 4 are considered. Path 1 is associated with a case where both the superelastic and shape memory effects are engaged ( $T = 20C < A_f$ ), but the heat generation phenomenon is neglected. Path 2 is similar to path 1 but includes effects of the heat generation on the transformation from the austenite to the martensite phase. In both cases, the energy dissipation region includes the entire region

beneath the stress-strain curve. Path 3 invokes the superelastic behavior. Therefore, it seems that paths 1 and 2 represent more dissipative conditions and for this reason, are chosen.



**Fig. 3.** A comparison between the time histories of the lateral deflection of the central point, predicted by present results and results of Tiberkak et al. [30].



Fig. 4.The considered loading paths of the SMA.

Three different velocities (V = 5, 10,  $30\frac{m}{s}$ ) and four different masses (m = 0.8, 2, 4, 6.8 kg) are considered for the indenter. The volume fractions of the composite (V<sub>c</sub>) and SMA wires (V<sub>s</sub>) in different layers are adopted as: V<sub>c</sub> = [0.5/1/1/1/0.5] and V<sub>s</sub> = [0.5/0/0/0/0/ 0.5], respectively. The thermal and mechanical specifications of the SMA wires are [31]:

$$\begin{split} \epsilon_{l} &= 0.067, \rho = 6450 \, \frac{\text{kg}}{\text{m}^{3}}, \text{E}_{A} = 67 \text{ GPa}, \text{E}_{M} \\ &= 26.3 \text{ GPa}, \text{G} = 29.4 \text{ GPa}, \text{A}_{\text{s}} = 34.5 \, ^{\circ}\text{C} \\ \text{A}_{\text{f}} &= 49 \, ^{\circ}\text{C}, \text{M}_{\text{s}} = 18.4 \, ^{\circ}\text{C}, \text{M}_{\text{f}} = 9 \, ^{\circ}\text{C}, \text{C}_{A} \\ &= 13.8 \, \frac{\text{MPa}}{^{\circ}\text{C}}, \text{C}_{M} = 8 \, \frac{\text{MPa}}{^{\circ}\text{C}}, \text{T}_{0} = 20 \, ^{\circ}\text{C} \end{split}$$

#### 4.3. Results of V=5m/s

The stress-strain variations and time histories of the martensite volume fraction and temperature rise of the impacted point of the plate are depicted in Figs. 5 to 7 for various indenter masses and compared with the traditional wherein the impact-induced heat results generation is neglected. From Fig. 5 it can be deduced that the highest stress induced by the 0.8kg mass is lower than that required for the martensite phase transformation. The dissipated energy includes the whole region above each stress-strain curve. The difference percent in the dissipated energy due to neglecting the heat generation is given in Table 1, for various initial velocities and masses of the indenter. As Fig. 5 shows, more energy dissipation is expected considering the heat when generation phenomenon. Based on the results of Table1, the maximum discrepancy due to neglecting the heat generation is associated with the 6.8 kg indenter (8.7%). Comparing Figs. 6 and 7, reveals that the temperature rise variations are somewhat in relation to the time history of the resulting martensite volume fraction. As may be easily noted from Figs. 5 to 7, heat generation phenomenon leads to higher stresses but less mechanical strains (due to the resulting relaxation) and consequently, slightly less martensite volume fraction. On the other hand, as may be expected, the contact time increases with the indenter mass.



**Fig. 5.** Effects of the impact-induced heat generation on variations of the stresses and strains of the impacted point of the plate, for various indenter masses (V=5m/s).



Fig. 6. Effects of the impact-induced heat generation on time history of the martensite volume fraction of the impacted point of the plate, for various indenter masses (V=5m/s).



Fig. 7. Effects of the impact-induced heat generation on time history of the temperature rise of the impacted point of the plate, for various indenter masses (V=5m/s).

**Table 1.** The discrepancies in the dissipated energy and the resulting stress magnitude, due to neglecting the impact-induced heat generation effects.

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_	Indenter velocity (m/s)	Indenter mass (kg)	Difference in the dissipated energy (%)	Maximum discrepancy in the stress (%)
	5	0.8	0	0
		2	0.86	0.1
		4	2.73	1.9
		6.8	8.73	3.9
	10	0.8	1.89	1.1
		2	10.17	5.1
		4	52.72	9.7
		6.8	40.40	11.8
	30	0.8	35.65	12.9
		2	28.82	24.3
		4	26.93	13.2
		6.8	22.72	11.9

#### 4.4. Results of V=10m/s

Effects of the resulting heat generation are illustrated in Figs. 8 to 10. In this case, the phase transformation occurs even for the 0.8kg indenter, but the martensite volume fraction is still below 100 percent, even for the 6.8kg indenter. As can be seen from Table 1, the discrepancies have increased by increasing the indenter velocity.



Fig. 8. Effects of the impact-induced heat generation on variations of the stresses and strains of the impacted point of the plate, for various indenter masses (V=10m/s).



Fig. 9. Effects of the impact-induced heat generation on time history of the martensite volume fraction of the impacted point of the plate, for various indenter masses (V=10m/s).



Fig. 10. Effects of the impact-induced heat generation on time history of the temperature rise of the impacted point of the plate, for various indenter masses (V=10m/s).

Lateral deflection contour at the instant of maximum contact force is illustrated in Fig. 11.



**Fig. 11.** Contours of the lateral deflection of the SMA composite plate, for the 6.8kg indenter.

#### 4.5. Results of V=30 m/s

Finally, results are extracted for the maximum initial velocity of the indenter, i.e., V=30m/s and are plotted in Figs. 12 to 14. These figures show that in this case, neglecting the heat generation may lead to unreliable results with a mean error of 20% as may be seen in Table 1. Figure12 shows that SMA has become saturated at the impacted point, for the 4 and 6.8kg indenters, as Fig. 13 confirms.



Fig. 12. Effects of the impact-induced heat generation on variations of the stresses and strains of the impacted point of the plate, for various indenter masses (V=30m/s).



Fig. 13. Effects of the impact-induced heat generation on time history of the martensite volume fraction of the impacted point of the plate, for various indenter masses (V=30m/s).



Fig. 14. Effects of the impact-induced heat generation on time history of the temperature rise of the impacted point of the plate, for various indenter masses (V=30m/s).

Figure 14 reveals that the temperature rise has reached up to 12°C. This amount of temperature rise cannot be neglected. However, since the material properties assumed to be temperatureindependent, their effects on the global responses have not been remarkable. This can be observed from Fig. 15 that illustrates effects of the heat generation on the contact force of the plate. On the other hand, results of Table 1 reveal that due to the nonlinear nature of the governing and contact equations, the resulting errors in the dissipated energy are casedependent and may reach up to 52% in some cases. Therefore, when the stress distribution is uniform and nonlocal, these effects may change the results by about 20%.



Fig. 15. Effects of the impact-induced heat generation on time history of the temperature rise of the impacted point of the plate, for various indenter masses (V=30m/s, m=6.8kg).

#### 5. Conclusions

In the present paper, effects of the impactinduced heat generation on impact responses and phase transformations of a hybrid SMA composite plate are investigated through proposing a refined Helmholtz free energy expression and refined constitutive and contact laws, in addition to employing a return-map Newton-Raphson method for enhancement of the numerical solution algorithm. Several comparative results are extracted to evaluate the amount of errors caused by neglecting the heat generation effects in the dissipated energy. Also, results of the temperature profiles revealed that while effects of the heat generation on the global behaviors (e.g., the contact force) may be small; these effects can be significant for components with temperature-dependent properties. Moreover, results showed that for the considered data, if the local and global responses are almost identical (e.g., for components under pure

extensive impact loads), the errors due to neglecting the heat generation effects may reach up to 50% in the computed dissipated energy.

## References

- [1] S. M. R. Khalili, A. Shokuhfar, and F. Ashenai Ghasemi, "Effect of smart stiffening procedure on low-velocity impact response of smart structures", *J. Mater Proc. Tech.*, Vol. 190, No. 1-3, pp. 142-152, (2007).
- [2] A. Shokuhfar, S. M. R. Khalili, F. Ashenai Ghasemi, K. Malekzadeh, and S. Raissi, "Analysis and optimization of smart hybrid composite plates subjected to low-velocity impact using the response surface methodology (RSM)", *Thin-Wall Struct*, Vol. 46, No. 11, pp. 1204-1212, (2008).
- [3] M. Meo, F. Marulo, M. Guida, and S. Russo, "Shape memory alloy hybrid composites for improved impact properties for aeronautical applications", *Compos. Struct.*, Vol. 95, pp.756-766, (2013).
- [4] E. H. Kim, I. Lee, J. H. Roh, J. S. Bae, I. H. Choi, and K. N. Koo, "Effects of shape memory alloys on low velocity impact characteristics of composite plate", *Compos. Struct.*, Vol. 93, No. 11, pp. 2903-2909, (2011).
- [5] J. H. Roh and J. H. Kim, "Adaptability of hybrid smart composite plate under low velocity impact", *Compos. Part B*, Vol. 34, No. 2, pp.117-125, (2003).
- [6] M. Shariyat and R. Jafari, "Nonlinear low-velocity impact response analysis of a radially preloaded two-directionalfunctionally graded circular plate: A refined contact stiffness approach", *Compos. Part B*, Vol. 45, No .1, pp. 981-994, (2013).
- [7] M. Shariyat and F. Farzan, "Nonlinear eccentric low-velocity impact analysis of a highly prestressed FGM rectangular plate, using a refined contact law", *Arch. Appl. Mech.*, Vol. 83, No.4, pp. 623-641, (2013).

- [8] M. Shariyat and F. Farzan Nasab, "Low-velocity impact analysis of the hierarchical viscoelastic FGM plates, using an explicit shear-bending decomposition theory and the new DQ method", *Compos. Struct.*, Vol. 113, No.1, pp. 63-73, (2014).
- [9] M. Shariyat and M. Moradi, "Enhanced algorithm for nonlinear impact of rectangular composite plates with SMA wires, accurately tracing the instantaneous and local phase changes", *Compos. Struct.*, Vol. 108, pp. 834-847, (2014).
- [10] M. Shariyat and S. H. Hosseini, "Accurate eccentric impact analysis of the preloaded SMA composite plates, based on a novel mixed-order hyperbolic global–local theory", *Compos. Struct.*, Vol. 124, pp. 140-151, (2015).
- [11] D. Helm and P. Haupt, "Shape memory behaviour: modeling within continuum thermo-mechanics", *Int. J. Solids Struct.*, Vol. 40, No. 4, pp. 827-849, (2003).
- [12] H. Tobushi and Y. Shimeno, T. Hachisuka, and K. Tanaka, "Influence of strain rate on super elastic properties of TiNi shape memory alloy", *Mech. Mater.*, Vol. 30, No. 2, pp. 141-150, (1998).
- [13] M. Kadkhodaei, RKND. Rajapakse, M. Mahzoon and M. Salimi, "Modeling of the cyclic thermo-mechanical response of SMA wires at different strain rates", *Smart Mater. Struct.*, Vol. 16, No. 6, pp. 2091-2101, (2007).
- [14] P. C. C. Monteiro, M. A. Savi, T. A. Netto, and P. M. C. Pacheco, "A Phenomenological description of the thermo-mechanical coupling and the rate-dependent behavior of shape memory alloys", J. Intell. Mater. Sys. Struct., Vol. 20, No. 14, pp. 1675-1687, (2009).
- [15] C. Morin, Z. Moumni and W. Zaki, "A constitutive model for shape memory alloys accounting for thermomechanical coupling", *Int. J. Plast.*, Vol. 27, No. 5, pp. 748-767, (2011).

- [16] J. H. Roh, "Thermomechanical Modeling of shape memory alloys with rate dependency on the pseudoelastic behavior", *Math. Prob. Eng.*, Vol. 20, No. 1, pp.41-65, (2014).
- [17] J. Ignaczak and M. Ostoja-Starzewski, *Thermoelasticity with finite wave speeds*, Oxford University Press, United States, New York, (2010).
- [18] M. R. Eslami, R. B. Hetnarski, J. Ignaczak, N. Noda, N. Sumi and Y. Tanigawa, *Theory of elasticity and thermal stresses*, Springer, (2013).
- [19] L. C. Brinson, "One dimensional constitutive behavior shape memory alloys: Thermo-mechanical derivation with non-constant material functions and redefined martensite internal variable", *J. Intell. Mater Syst. Struct.*, Vol. 4, No. 2, pp. 229-242, (1993).
- [20] D. C. Lagoudas, *Shape Memory Alloys: Modeling and Engineering Applications*, Springer, (2008).
- [21] J. N. Reddy, Mechanics of Laminated Composite Plates and Shells-Theory and Analysis, CRC Press, Boca Raton, (2003).
- [22] V. Birman, "An approach to optimization of shape memory alloy hybrid composite plates subjected to low-velocity impact", *Composites: Part B*, Vol. 27, No. 5, pp. 439-446, (1996).
- [23] J. R. Turner, "Contact on a transversely isotropic half-space, or between two transversely isotropic bodies", *Int. J. Solids Struct.*, Vol. 16, No. 5, pp. 409-19, (1980).
- [24] A. Niknami and M. Shariyat, "Refined constitutive, bridging, and contact laws for including effects of the impact-

# How to cite this paper:

A. Niknami and M. Shariyat, "Influence of heat generation on the phase transformations and impact responses of composite plates with embedded SMA wires", *Journal of Computational and Applied Research in Mechanical Engineering*, Vol. 6. No. 2, pp. 13-26

DOI: 10.22061/jcarme.2017.581

URL: http://jcarme.srttu.edu/? action=showPDF&article=581

induced temperature rise in impact responses of composite plates with embedded SMA wires", *Thin-Walled Struct.*, Vol. 106, pp. 166-178, (2016).

- [25] S. H. Yang and C.T. Sun, "Indentation law for composite laminates", *In: Composite materials: testing and design* (6th conference), ASTM STP-787, pp. 425-49, (1982).
- [26] M. R. Eslami, *Finite elements methods in mechanics*, Springer, (2014).
- [27] E. Serra and M. Bonaldi, "A finite element formulation for thermoelastic damping analysis", *Int. J. Numer. Meth. Eng.*, Vol. 78, No. 6, pp. 671-691, (2009).
- [28] M. Shariyat and A. Niknami, "Impact analysis of strain-rate-dependent composite plates with SMA wires in thermal environments: Proposing refined coupled thermoelasticity, constitutive, and contact models", *Compos. Struct.*, Vol. 136, pp. 191-120, (2016).
- [29] M. Shariyat and A. Niknami, "Layerwise numerical and experimental impact analysis of temperature-dependent transversely flexible composite plates with embedded SMA wires in thermal environments", *Compos. Struct.*, Vol. 153, pp. 692–703, (2016).
- [30] R. Tiberkak, M. Bachene, S. Rechak and B. Necib, "Damage prediction in composite plates subjected to low velocity impact", *Compos. Struct.*, Vol. 83, No. 1, pp.73-82, (2008).
- [31] L. C. Brinson and R. Lammering, "Finite element analysis of the behavior of shape memory alloys and their applications", *Int. J. Solids Struct.*, Vol. 30, No. 23, pp.3261-3280, (1993).

